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Lecture 39 Review

Chap. 1 Basic principles

Can separate center of mass and internal motion

$$T = T_{cm} + T'$$

$$T_{cm} = \frac{1}{2} M V_{cm}^2$$

$$T' = \sum_i \frac{1}{2} m_i \dot{r}'_i{}^2$$

$$L = L_{cm} + L'$$

$$L_{cm} = MR \wedge V$$

$$L' = \sum_i m_i r'_i \wedge v'_i$$

Central force motion $V(\vec{r}) = V(r)$

$$E = \frac{1}{2} m \dot{r}^2 + V_{eff}(r)$$

$$V_{eff} = V(r) + l^2 / 2mr^2$$

Inverse-square law gives ellipses

$$E = -MmG/2a$$

a = semi-major axis

Chapter 2 Accelerated coordinate systems

$$\left(\frac{d}{dt}\right)_{inertial} = \left(\frac{d}{dt}\right)_{body} + \omega \wedge r$$

$$\left(\frac{dV}{dt}\right)_{inertial} = \left(\frac{dV}{dt}\right)_{body} + \omega \wedge r \wedge \nabla V$$

Chapter 3 Lagrangian Dynamics

Incorporate forces of constraint
and use generalized coordinates

Holonomic constraints

$$f_j(x_1, \dots, x_n, t) = c_j \quad j=1, 2, \dots, k$$

x_1, \dots, x_n = "real" coordinates

Generalized coordinates q_i

$$x_j = x_j(q_1, \dots, q_{n-k}, t) \quad j=1, \dots, n$$

Lagrangian

$$L = T - V = L(q_1, \dots, q_{n-k}, \dot{q}_1, \dots, \dot{q}_{n-k}, t)$$

Lagrange's eq.

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{q}_s} - \frac{\partial L}{\partial q_s} = 0 \quad s=1, \dots, n-k$$

Hamilton's principle

$$\delta \int_{t_1}^{t_2} L[q(t), \dot{q}(t), t] dt = 0$$

$$\delta q(t_1) = \delta q(t_2) = 0$$

gives Lagrange's eq.

4 Small Oscillations

Expand about equil.

$$-\left. \frac{\partial V}{\partial q_\sigma} \right|_{q_0} = 0$$

$$\sigma = 1, \dots, n$$

$$q_0 = q_0^c + \eta_0$$

$$\dot{q}_0 = \dot{\eta}_0$$

$$T = \frac{1}{2} \sum_{\sigma, \lambda} m_{\sigma\lambda} \dot{\eta}_\sigma \dot{\eta}_\lambda$$

$$m_{\sigma\lambda} = \sum_i m_i \left. \frac{\partial x_i}{\partial q_\sigma} \right|_{q_0} \left. \frac{\partial x_i}{\partial q_\lambda} \right|_{q_0}$$

$$V = V_0 + \frac{1}{2} \sum_{\sigma, \lambda} V_{\sigma\lambda} \eta_\sigma \eta_\lambda$$

$$V_{\sigma\lambda} = \left. \frac{\partial^2 V}{\partial q_\sigma \partial q_\lambda} \right|_{q_0}$$

$$\sum_\lambda (m_{\sigma\lambda} \ddot{\eta}_\lambda + V_{\sigma\lambda} \eta_\lambda) = 0 \quad \sigma = 1, \dots, n$$

Normal modes

$$z_\sigma = z_\sigma^0 e^{i\omega t}$$

$$\eta_\sigma = \text{Re } z_\sigma$$

$$\sum_\lambda [V_{\sigma\lambda} - \omega^2 m_{\sigma\lambda}] z_\lambda^0 = 0$$

Eigenvalues

$$\det | \underline{V}_{\sigma s} - \omega^2 m_{\sigma s} | = 0$$

Eigenvectors

$$(\underline{V} - \omega^2 \underline{m}) \underline{f}^{(s)} = 0$$

$$\underline{\eta}(t) = \sum_s C^s \underline{f}^{(s)} \cos(\omega_s t + \phi_s)$$

Chap. 5 Rigid Bodies

$$T = \frac{1}{2} \sum_{i,j=1}^3 I_{ij} \omega_i \omega_j$$

$$L_i = \sum_j I_{ij} \omega_j$$

IF body is not symmetric then
 $\vec{\omega}$ and \vec{L} can point
in different directions

$$I_{ij} = \int d^3r \rho(r) (\delta_{ij} r^2 - r_i r_j)$$

Hamiltonian Dynamics

$$p_\sigma \equiv \frac{\partial L}{\partial \dot{q}_\sigma}$$

$$H \equiv \sum_\sigma p_\sigma \dot{q}_\sigma - L$$

$$L = L(q, \dot{q}, t)$$

$$H = H(p, q, t)$$

$$\frac{\partial H}{\partial p_\alpha} = \dot{q}_\alpha$$

$$\frac{\partial H}{\partial q_\alpha} = -\dot{p}_\alpha$$

Lagrange's equations are invariant under transformation of generalized coordinates

Hamilton's equations are invariant under much broader class of canonical transformations

Transformation of Hamilton's eq. is canonical if F_0 form is unchanged

F_0 , example can mix up coordinates and momenta

Poisson Brackets

$$[F, G]_{PB} = \sum_\alpha \left(\frac{\partial F}{\partial q_\alpha} \frac{\partial G}{\partial p_\alpha} - \frac{\partial F}{\partial p_\alpha} \frac{\partial G}{\partial q_\alpha} \right)$$

$$\frac{dF}{dt} = - [H, F]_{PB} + \frac{\partial F}{\partial t}$$

Canonical quantization procedure

In QB replace

$$[A, B]_{PB} \rightarrow \frac{1}{i\hbar} [\hat{A}, \hat{B}]$$

$$\frac{d\hat{F}}{dt} = \frac{i}{\hbar} [\hat{H}, \hat{F}] + \partial \hat{F} / \partial t$$

Strings

$$L = \int_0^l \mathcal{L} dx$$

$$\mathcal{L} = \frac{1}{2} \sigma(x) \left(\frac{\partial U}{\partial t} \right)^2 - \frac{1}{2} \tau(x) \left(\frac{\partial U}{\partial x} \right)^2$$

Eigenfunctions satisfy Sturm Liouville eq.

$$-\frac{\partial}{\partial x} \tau \frac{d\rho}{dx} + v(x)\rho = \omega^2 \sigma \rho$$

Eigenfunctions are orthogonal

$$\int_a^b \rho_p(x) \rho_q(x) dm = \delta_{pq}$$

$$dm = \sigma(x) dx$$

$$U(x,t) = \rho(x) \cos(\omega t + \phi)$$

Eigenfunctions are complete

Variational Principle

$$\omega^2[\rho] = \frac{\frac{1}{2} \int_a^b [\tau(x) (\rho'/x)^2 + v(x)\rho^2] dx}{\frac{1}{2} \int_a^b [\sigma(x) \rho^2] dx}$$

ok
good

guess
guess

f_a
 f_c

ρ_1
 ρ_2

generates

$$\omega^2[\rho] \geq \omega^2[\rho_1]$$

Green's Functions

Consider inhomogeneous Sturm Liouville
eq

$$-\frac{d}{dx} \left(\tau \frac{du}{dx} \right) + v u - \omega^2 \sigma u = \sigma(x) f(x)$$

To calculate response of system
to driving force $f(x)$

Introduce $(L_\omega - \omega^2 \sigma) G_\omega(x, y) = \delta(x-y)$

$$u(x) = \int_a^b G_\omega(x, y') \sigma(y') f(y') dy'$$

$$G_\omega = \sum_{n=1}^{\infty} \frac{p_n(x) p_n(y)}{\omega_n^2 - \omega^2}$$

or

$$= - \frac{u_1(x_<) u_2(x_>)}{}$$

Perturbation Theory

$$p_n(x) = p_n(x) + \epsilon \sum_{q \neq n} p_q(x) \frac{1}{\omega_n^2 - \omega_q^2} \langle q | v | n \rangle + O(\epsilon^2)$$

$$\omega^2 - \omega_n^2 = \epsilon \langle n | v | n \rangle + O(\epsilon^2)$$